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# Türkiye'de Soğuk Zincir Lojistiği ve Soğuk Zincirde İklim Dostu Teknolojiler/Alternatifler



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#### Average temperature anomaly, Global

Global average land-sea temperature anomaly relative to the 1961-1990 average temperature in degrees celsius (°C). The red line represents the median average temperature change, and grey lines represent the upper and lower 95% confidence intervals.



Our World in Data





\* Source: European Commission









SOURCE: Green Cooling Technologies Market trends in selected refrigeration and air conditioning subsectors . Deutsche Gesellschaft für Internationale Zusammenarbeit (GIZ) GmbH





Figure 8: Comparison of allowed emissions according to phase-down in the F-Gas regulation and the expected refrigerant demand (Clodic et al., 2010). Blue circles indicate the average GWP that matches both the demand and

the phase-down.

SOURCE: Green Cooling Technologies Market trends in selected refrigeration and air conditioning subsectors . Deutsche Gesellschaft für Internationale Zusammenarbeit (GIZ) GmbH





#### Share of global cumulative CO2 emissions, 1793

Our World in Data

Each country or region's share of cumulative global carbon dioxide (CO<sub>2</sub>) emissions. Cumulative emissions are calculated as the sum of annuals emissions from 1751 to a given year.



2017

CHART

MAP

DATA



1751



#### Share of global cumulative CO<sub>2</sub> emissions, 2017

Each country or region's share of cumulative global carbon dioxide (CO<sub>2</sub>) emissions. Cumulative emissions are calculated as the sum of annuals emissions from 1751 to a given year.













Annual share of global CO<sub>2</sub> emissions



Each country's share of global carbon dioxide (CO2) emissions. This is measured as each country's emissions divided

Source: Our World in Data based on Global Carbon Project (2018)





Our World in Data

Refrigerant	Critical temperature (°C)	Critical pressure (bar)	Ozone depletion potential	Global warming potential (100 years)	Flammable or explosive	Toxicity
R12	100.9	40.6	0.9	8100	No	No
R22	96.2	49.8	0.055	1500	No	No
R32	78.4	58.3	0	650	Yes	No
R134a	101.1	40.7	0	1200	No	No
R152a	113.5	45.2	0	140	Yes	No
R404A	72.1	37.4	0	3300	No	No
R407C	86.8	46.0	0	1600	No	No
R410A	72.5	49.6	0	1900	No	No
Propane (R290)	96.8	42.5	0	3	Yes	No
Isobutane (R600a)	135.0	36.5	0	3	Yes	No
Ammonia (R717)	132.2	113.5	0	0	Yes	Yes
Carbon dioxide (R744)	31.0	73.8	0	1	No	No

CFC, chlorofluorocarbon; HCFC, hydrochlorofluorocarbon; HFC, hydrofluorocarbon





#### CO2 (R744) Refrigerant

#### Table 5. Comparison of R-744 with other refrigerants

	R-744	HFOs	HCs	R-717
Capacity				
Efficiency				
Pressure				
Environmental impact				
Flammability				
Toxicity				
Availability of refrigerant				
Availability of components				
Availability of expertise				
Cost of refrigerant				
Cost of system				



Refrigerant is similar to HFCs;

Aspect of the refrigerant is worse than HFCs; Aspect of the refrigerant is better than HFCs. 11



#### **Refrigerants Comparison**



Figure 3: Latent beat of vaporization/condensation for selected refrigerants.

Figure 4 shows the values for the volumetric refrigeration effect of the selected refrigerants. As can be seen in the plot  $CO_2$  has values which are





#### **Refrigerants Comparison**





Figure 4: Volumetric refrigeration capacity (complete evaporation) for selected refrigerants.



# **Refrigeration Cycle**

#### **Cold Storage Refrigeration**







# **Refrigeration Cycle**

#### **Refrigeration Cycle**







#### **Refrigerated Truck**







#### Energy Consumption for refrigerated road transport

- Approximately 650000 refrigerated road vehicles are currently in use within the EU. These vehicles can be grouped into three main types:
- Small converted vans (up to 3.5 tonnes, for example for catering or ice cream distribution),
- Rigid vehicles (trucks, up to 32 tonnes) and
- Articulated vehicles (up to 44 tonnes), which are used for the majority of refrigerated road transportation operations.
- Food transport estimated to be responsible for 1.8 % of total emissions.





#### Refrigerated Truck Energy Consumption

Body	Minimum		Fuel consumption		Associated refrigeration	
Length/Volume/Type	refrigeration duty		(l/hr)		Duty (W)	
	(W) for AT	TP based				
	on 0.4 W/m <sup>2</sup> K					
	-20 °C	0 °C	-20 °C	0 °C	-20 °C	0 °C
6 m/ 30 m <sup>3</sup> / Rigid Lorry	2380	1428	1.5	2.0	3000	5000
<9 m/ 30 m <sup>3</sup> / Rigid	3850	2310	2.5	3.0	6000	9000
Lorry						
13.6 m/90 m <sup>3</sup> /Rigid	5250	3150	3.0	4.0	7500	13500
Semi Trailer						
13.6 m/>90 m3/Rigid	-	-	4.5	5	9500	17500
Semi Trailer						





#### Vapour Compression Refrigeration Unit



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#### Absorption Refrigeration utilising exhaust heat







#### Eutectic systems

- Phase Change Materials
- Combined Phase Change Materials and Vapor Compression System







#### **Cryogenic Cooling Systems**

- Liquid Nitrogen or Carbon Dioxide Injection
- Mechanical Systems







Figure 5. BOC 'Polarstream' Liquid Nitrogen Cooling System

#### Air Cycle Refrigeration

Transport refrigeration has been identified as a potential application area for air cycle systems on the grounds of weight, robustness, leakage, reliability and maintenance. Air cycle systems are also less sensitive to part-load operation.







#### Solar Powered Transport Refrigeration







#### Natural Refrigerant System Types

- Low-temperature (LT) or medium-temperature (MT) CO2 overfeed
- MT HFC DX with LT CO2 DX cascade
- HFC DX primary over combined MT overfeed with LT CO2 DX
- NH3-flooded primary over combined MT overfeed with LT CO2 DX
- CO2 transcritical booster system
- Self-contained water-cooled hydrocarbon





#### <mark>Tek kademeli basit transkritik çevrim</mark>











Fig. 3.1.1: Pumped system

Fig. 3.1.2: DX system

Fig. 3.1.3: Combined system





#### <mark>Tek kademeli sıvı/gaz <u>ayırıcılı transkritik</u> çevrim</mark>















• CO2 (R744) Ejector Integration



Figure 3.8 Schematic of R744 vapour compression refrigeration cycle with a two-phase ejector. Adapted from Sumeru et al.  $(2012)_{30}$ 





• CO2 (R744) Ejector Integration



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Figure 3.9 P-h diagram of R744 vapour compression refrigeration cycle with a two-phase ejector and comparison with standard cycle. Adapted and modified from Sumeru et al. (2012)



• CO2 (R744) Ejector Integration







• CO2 (R744) Cascade System







• CO2 (R744) Booster System







• CO2 (R744) Booster System



Figure 3.6 Transcritical booster system in P-h layout. Adapted from Ge & Tassou (2011)





- Supermarkets are energy intensive buildings consuming 3-4% of the total annual electricity in industrialized countries
- 35-50% of this total electricity is consumed in the supermarket refrigeration systems.
- Supermarket refrigeration systems are one of the largest consumers and emitters of high GWP refrigerants; 30% of Europe HFC consumption 22% annual leakage rate.





• CO2 (R744) Booster System









CO2 (R744) Transritical System



Figure 2: Year 2013 map of CO<sub>2</sub> trans-critical booster systems in Europe (top-left), CO<sub>2</sub> cascade systems in Europe (bottom-left) and CO2 trans-critical and cascade/secondary stores in world (right) (Shecco, 2014)





• CO2 (R744) Transritical System





Figure 3: Systems schematics A) HFC reference systems RS1-RS2-RS3. B) TR1-Parallel transcritical. C) TR2-transcritical booster + Parallel medium temperature cycle. TR3- has only transcritical booster units D) TR4 and TR5-transcritical booster with flash gas by-pass.



• CO2 (R744) Transritical System







• CO2 (R744) Transritical System

Fig. 17. "All-in-one" transcritical R744 booster supermarket refrigeration system equipped with multi-ejector rack (IESPC unit) (Hafner et al., 2016).







• Self Contained HC Rerigeration System







• Self Contained HC Rerigeration System





Figure 3-6. Self-contained hydrocarbon condensing units with a hydronic loop





Figure 3-1. LT or MT CO2 overfeed system

























# THANKS FOR YOUR ATTENTION!



